Design of non-linear controller of Rotary Inverted Pendulum

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1 Introduction

The Rotary Inverted Pendulum is a highly nonlinear system. A rotary inverted pendulum system (see Figure 1), has a rotating arm, which is driven by a motor with a pendulum mounted on its rim. The pendulum moves as an inverted pendulum in a plane perpendicular to the rotating arm. The controller needs



Figure 1: Rotary Inverted Pendulum

to stabilize two angles namely, θ -the angle of the motor joint and α -the angle of the pendulum. Now, θ has to be stabilized to be within a particular bound while α has to be made 180 °.

2 Dynamics of the System

In this section the dynamics of the Pendulum is given. The system parameters were taken from the simulink model provided.

Parameter	Description	Value
J_{eq}	Equivalent moment of inertia about motor shaft .	$1.23 \times 10^{-4} \text{kg-m}^2$
m	Mass of the pendulum assembly	0.027kg
r	Length of arm pivot to pendulum pivot	0.08260m
J_m	Motor shaft moment of inertia	0.00011kg-m ²
L	Total length of pendulum.	0.191m
B_{eq}	Arm viscous damping	$0.0015 \frac{N-m}{rad/s}$
g	gravitational constant	9.81 m/s^2
K_m	Motor back-electromotive force constant	$0.02797 \ \frac{V}{rad/s}$
K _t	Motor torque constant	0.02797N-m
\mathbf{R}_m	Motor armature resistance.	$3.30 \ \Omega$

Using the values of Table(1) following constants were calculated.

$$a = J_{eq} + J_m + mr^2$$

$$b = mLr$$

$$c = \frac{4mL^2}{3}$$

$$d = mgL$$

$$G = \frac{K_t K_m + B_{eq} R_m}{R_m}$$

$$(1)$$

The nonlinear state equation of the system is given below:

$$\dot{x_1} = x_2 = \dot{\alpha} \tag{2}$$

$$\dot{x_2} = F_1(X) + G_1(X)u \tag{3}$$

$$\dot{x_3} = x_4 = \dot{\theta} \tag{4}$$

$$\dot{x_4} = F_2(X) + G_2(X)u \tag{5}$$

where F_1 , F_2 , G_1 , G_2 are:

$$F_1(X) = \frac{-0.5b^2 \sin(2x_1)x_2^2 - Gbx_4 \cos(x_1) + adsin(x_1)}{ac - b^2 \cos^2 x_1} \tag{6}$$

$$G_1(X) = \frac{K_t b cos(x_1)}{R_m (ac - b^2 cos^2 x_1)}$$
(7)

$$F_2(X) = \frac{cF_1(X) - dsin(x_1)}{bcos(x_1)}$$
(8)

$$G_2(X) = \frac{K_t c}{R_m (ac - b^2 cos^2 x_1)}$$
(9)

$$T = \frac{(u - K_m \dot{\theta}) K_t}{R_m} \tag{10}$$

3 Sliding Mode Controller

The sliding mode controller for the inverted pendulum is designed as described in [1]. Here two sliding surface are considered for sliding mode control. They are defined below:

$$s_1 = x_2 + \lambda_1 x_1 \tag{11}$$

$$s_2 = x_4 + \lambda_3 x_3 \tag{12}$$

The Lyapunov function is taken as:

$$V = |s_1| + \lambda_2 |s_2| \tag{12}$$

To guarantee the stability of the feedback system, the control signal u is taken such that

$$\dot{V} = -k \times sat(\frac{V}{\Phi})$$

where saturation function is defined as

$$sat(\frac{V}{\Phi}) = \frac{V}{\Phi} \qquad if \quad \Phi < |V| \tag{13}$$

$$sat(\frac{V}{\Phi}) = sign(V)$$
 otherwise (14)

The final control signal is shown below:

$$u = \frac{-ksat(\frac{V}{\Phi}) - (\lambda_1 x_2 + F_1(X))sign(s_1) - \lambda_2(\lambda_3 x_4 + F_2(X))sign(s_2)}{G_1(X)sign(s_1) + \lambda_2 G_2(X)sign(s_2)}$$
(14)

4 Result

With this sliding mode controller with two sliding surfaces, there are five constants $\lambda_1, \lambda_2, \lambda_3, k, \Phi$. The values of the constants are given in Table 2

Table 2: Values used for modeling

Constant	Value		
λ_1	0.2		
λ_2	0.9		
λ_3	0.9		
k	20		
Φ	0.5		



Figure 2: variation of θ and α with initial disturbance $\alpha = 25^{\circ}$

The results achieved for the initial value of α being 27 ° is shown in Figures(2). The above figures show that the controller is able to stabilize both θ and α . We can see the prevailing chattering that is present in our output. The α has a maximum chattering of 5°. This controller is able to stabilize the inverted pendulum with an initial α in the range of 27°. Now we can reduce chattering to 3° by changing the constants, but that will reduce the maximum range of α for

which this controller can stabilize the system.

References

 Khanesar, Mojtaba Ahmadieh, Mohammad Teshnehlab, and Mahdi Aliyari Shoorehdeli. "Sliding mode control of rotary inverted pendulm." Control & Automation, 2007. MED'07. Mediterranean Conference on. IEEE, 2007.