# FAST PLANAR LINKAGE MECHANISM SOLVER AND AUTOMATIC DIFFERENTIATION FOR GRADIENT-BASED OPTIMIZATION AND PATH SYNTHSIS

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51. Introduction. The longstanding challenge in engineering design has been the analysis and synthesis of various kinematic mechanisms, a topic that has engaged 6 countless engineers and scientists over time [17]. The understanding of the design 7 of intricate kinematic systems is still elusive, and typically is restricted to certain 8 9 tasks. The design process often relies on a combination of trial and error, specialized knowledge, and heuristics to discover effective designs. However, the emergence of 10computational design strategies in recent years has prompted a shift towards inverse 11 kinematic design via optimization techniques [17, 3, 4, 28, 11, 1, 13, 19, 5, 7, 29, 8, 9, 122, 21, 14]. 13

14In the realm of computational approaches, the application of data-driven strategies for inverse design has garnered increasing interest. This trend has spurred a surge 15 in research centered on the utilization of statistical machine learning and deep learning models in inverse kinematics [29, 8, 9, 14, 12] and in engineering design more gener-17 ally [22]. Yet, the effectiveness of many computational and data-driven methodologies 18 19 is hindered by the computational power required for mechanism simulations. Some methods demand millions of simulations for efficient inverse kinematic design [12], 20 and others are confined to certain types of basic mechanisms, like 4-bars or 6-bars, 21 among others [4, 28, 11, 1, 13, 19, 5, 7, 29, 8, 9, 14]. 22

The primary constraint stems from the vastness of the design space – the sheer variety of problem requirements and mechanism variations is virtually boundless, which makes the exploration of more sophisticated mechanisms a daunting task, potentially necessitating billions or even trillions of simulations. Therefore, to facilitate the investigation of this immense design space for inverse kinematic synthesis, it is crucial to have computational tools equipped with faster solvers.

In this work we focus on the problem of kinematic synthesis of planar linkage 2930 mechanisms, however, unlike most of the work done more recently, we do not limit our work to specific topologies of mechanisms (such as 4-bars, 6-bars, etc.) and go beyond 31 the well understood and further investigated mechanisms. By exploring a wide range 32 of simple and complex 1-DOF (degrees of freedom) planar linkage mechanisms, we 33 attempt to provide a more general picture of the problem and address the limitations 34 35 at the most generalizable scale possible. In these kinds of mechanisms, the problem of inverse kinematic synthesis can have different types of goals, such as coupler path 36 synthesis, motion generation, and signal generation [18]. The path synthesis problem 37 can be described as the problem of designing linkage mechanisms that can generate 38 a particular path that is described by a finite series of point coordinates. Motion 39 generation can be thought of as the generalized version of the path synthesis where 40aside from point coordinates, the orientation of an attached rigid body (such as a 41 robot arm) is also prescribed. Finally, the function generation problem refers to the 42 43 problem of generating a specific series of output crank angles (or slider positions) at given angles (or positions) at the actuator, essentially transforming the signal from 44 the actuator (angle or position) to a different signal at the output crank (or slider). 45 See Figure 1 for more details. This work is created with a primary focus on the "Path 46 Synthesis" problem, although with some tweaking the findings of this work could be 47

- 48 adapted to be used for other types of problems such as "Function Generation" and
- 49 "Motion Generation" [18].



FIG. 1. Different types of problems in inverse kinematics of planar linkage mechanisms. Note that the yellow arm is the actuator arm

50As mentioned before, one of the main limiting factors for computational approaches in inverse kinematic design is the need for a significant number of simulations. In fact the current state-of-the-art method for path synthesis uses reinforcement learning (RL) for path synthesis, however the RL agent needs to be retrained for each target curve to find a solution [12]. In this work, the authors require 2 mil-54lion simulations of mechanisms with up to 11 joints for every target curve. Other approaches based on genetic programming [17] and genetic algorithms have a similar 56 limitation since they require even larger amounts of simulations to find feasible solutions for path synthesis problems sometimes requiring billions of simulations. This 58 can be rather time-consuming using conventional solvers currently being used by most 59researchers. Given this, in this work, we develop highly optimized solvers for planar 60 linkage mechanisms based on the solver proposed by Bächer *et al.* [3]. We take this 61 solver and optimize the code for the solution by vectorizing the process for multiple 62 timesteps and vectorizing the process for multiple timesteps and multiple mechanisms 63 simultaneously to take advantage of the highly optimized underlying linear algebra 64 packages in Julia. Furthermore, we further speed up the already accelerated solver 65 even more by implementing GPU-based solvers for the problem which speed up the 66 process even more. We demonstrate that these optimizations provide multiple orders 67 of magnitude improvement over a naive solver and speed up the simulation process 68 massively. We hope that with the advent of these faster solvers even existing methods 69 can be accelerated massively leading to faster or better optimizers that not only per-7071 form inverse kinematic synthesis faster but are also capable of exploring more complex mechanisms. We detail our work and the implementation of the fast solvers in the 72 73 sections that follow.

Finally, we take advantage of the automatic differentiation tools developed for Julia, specifically the ForwardDiff.jl package, to develop a gradient-based optimizer for planar linkage mechanisms to allow for the refinement of planar linkage mechanisms for path synthesis problems. The goal of this optimizer is to take candidate solutions

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that are capable of producing paths very similar to a given target path and adjust 78 79the mechanism to achieve as close a match to the target curve as possible. Existing gradient-based methods [3] are limited to exceedingly minor changes when it comes 80 to path matching because these methods introduce a penalty to the objective of the 81 solver to prevent mechanisms from falling into a locking configuration, however, this 82 makes the optimizer limited to more smooth paths and paths that are produced by 83 mechanisms which are sufficiently away from locking configuration, as otherwise the 84 penalty factor in the optimizer becomes the dominating factor in the gradient [3]. 85 In this work, we propose a novel approach for dynamically changing the weight of 86 the penalty term in the optimizer's objective to enable more effective gradient-based 87 optimization for planar linkage mechanisms. the details of how we achieve this are 88 provided in the sections that follow. 89

90 2. Background & Related Works. In this section, we will discuss some of the
 91 prior works which we utilize in our approach and discuss the overall topic of kinematic
 92 synthesis briefly.

93 2.1. Computational Inverse Kinematics. Computational approaches in the 94 inverse kinematics problem fall into three primary categories: a) Numerical-atlas ap-95 proaches, b) Optimization-based approaches, and c) Data-driven approaches. Below 96 we briefly discuss each approach in the context of how our work can benefit each 97 category of approaches.

Numerical atlas-based approaches:. The initial strategy involves building a reposi-98 99 tory of mechanisms, which subsequently generate paths that function like a "numerical atlas". This atlas can be used to find the nearest paths to any designated path and 100 employ the corresponding mechanism from the current database as a solution. Addi-101 tionally, integrating this retrieval step with local optimization of the mechanism can 102 bring us closer to the desired path [19, 6, 26]. In the majority of cases, the numerical 103atlas is generally restricted to a particular mechanism or a few types of mechanisms, 104105 such as a four-bar or six-bar mechanism. This is primarily due to the computational constraints of simulating numerous complex mechanisms, and the creation of a large 106 numerical atlas of intricate mechanisms would require excessive simulations. Search-107 ing for viable mechanisms in an expansive design space entails simulating countless 108 mechanisms, only to discover that they are locked or degenerate and thus unsuitable 109for inclusion in the numerical atlas. As a result, these methods often only apply to 110a limited range of simpler mechanisms. Furthermore, these basic mechanisms, with 111 a few joints, can only generate a limited variety of paths. For instance, it is known 112that a four-bar mechanism can precisely match at most five points of a path (and 113 even this is not always feasible) [23]. This demonstrates that even with a substantial 114115 atlas, the array of feasible paths that can be traced is limited. However, our work can address this issue by creating highly efficient solvers that facilitate the creation 116 of extensive numerical atlas databases, accommodating a higher level of complexity 117 and more joints. 118

Optimization-based approaches:. The second computational strategy is referred 119 to as the optimization-based approach. This category encompasses various works 120that utilize a range of optimization algorithms to determine the most appropriate 121 122mechanisms for a specific target path. While some scholars employ genetic algorithms or genetic programming techniques to produce mechanisms that can follow desired 123 paths [17, 13], others opt for optimization using Fourier descriptors [27, 31]. An 124additional approach involves gradient-based optimization [3] to modify an existing 125126 machine, although this method requires an already approximate solution to yield any

benefits. Apart from a few exceptions [17, 3], most of these methods are limited 127 to altering existing mechanisms through optimization [3], or they are limited to a 128 specific subset of problems. For example, Lipson et al. [17], focused on solving the 129straight-line problem using genetic programming. The performance of population-130 based optimization approaches, such as genetic algorithms, is mostly reliant upon 131 the volume of simulations that can be done in practical time. That is the size of 132 the populations and the number of generations the algorithm can be iterated over 133 will largely determine the level of success these approaches will have. And with 134 conventional solvers, the volume of simulation is highly limited. In our work, we 135 develop extremely fast solvers which potentially increase the volume of simulations 136by multiple orders of magnitude allowing for a much larger size of population and more 137 138 generations to be evaluated within the same amount of time which can hugely improve the performance of such approaches. Beyond this, we also develop a gradient-based 139optimization scheme that can refine mechanisms with better performance compared 140 to the existing state-of-the-art gradient-based methods such as the one proposed in [3]. 141

Data-driven approaches:. In recent times, the popularity of machine learning-142based methods has surged, leading to a number of published works that adopt a 143 144 data-driven perspective. Most of these studies incorporate the previously mentioned "numerical atlas" and optimization strategies within data-driven frameworks. For 145instance, Deshpande and colleagues combined the numerical atlas method with opti-146 mization in their research [7, 8, 9]. They utilize variational autoencoders (VAEs)[15] 147and clustering-based search techniques to identify suitable candidates capable of gen-148 149 erating a desired coupler curve. In their subsequent studies, they apply VAEs and conditional VAEs[25] to synthesize mechanisms. 150

The datasets employed in such studies are typically small and restricted to certain types of mechanisms (such as four-bar, six-bar, etc.). For example, a dataset of 6818 linkage mechanisms is used in [10]. These models could substantially benefit from larger datasets comprising millions of mechanisms. Other data-driven studies have attempted to generate mechanisms conditioned on paths [29], but these are again limited to four-bar mechanisms.

In contrast to these "numerical atlas" adaptations, some researchers have en-157deavored to translate the optimization approach into a machine learning framework. 158One such study applied deep Q learning [20] and Lipson's T and D operators [2]. 159Although these reinforcement learning (RL) based approaches are not confined to 160specific mechanisms, they require retraining for each new target shape. More re-161 cently, Fogelson *et al.*, proposed a new RL-based approach that was able to beat all 162existing approaches [12] in accuracy, however, as mentioned before the limitation of 163 these approaches is that they need retraining for every new target shape and specif-164165ically in this latest work each new target requires 2 million simulations to be done. 166 What is evident is that machine learning approaches show great promise, however, at the moment, the same limitations that we saw in optimization and numerical atlas 167 approaches can be observed in this category of approaches as well. As such our work 168 can significantly benefit this type of approach as well. 169

**2.2. Simulation of Kinematics.** There has been substantial work done in solving 1-DOF mechanisms, however, as mechanisms get more complex, solving them becomes costly and the complexity of the closed-form analytical equations becomes gargantuan. As a result, algorithms-based and numerical approaches to solving such systems are typically employed [30, 24]. The literature on this topic is extensive and beyond the scope of this report, however, there are a few relevant works that

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we will discuss here as they set up the context for future discussion. Broadly, two 176 177different approaches can be considered in simulating mechanisms beyond analytical approaches [24]. One approach is the numerical approach to solving kinematic sys-178 tems. An example of such an approach is Lipson's simulator, used for the genetic 179programming approach in [16]. In most numerical approaches planar mechanisms 180 are solved using numerical algorithms used for solving systems of non-linear equa-181 tions (such as Newton-Raphson or Broyden's method), these approaches are capable 182 of simulating very complex systems, however, in many complex systems the solution 183 is not unique and these simulators only produce one of the possible results [30, 24]. 184 Despite this, these numerical approaches are the most general solvers that can handle 185all types of mechanisms which graphical approaches might not be capable of, however, 186 187 in the context of inverse kinematic design this matters less as we see. The other approach to solving linkage mechanisms is to take a graphical approach and solve planar 188 mechanisms from a purely kinematic approach. One such simulation approach which 189 focuses purely on kinematics is the one proposed by Bächer et al. [3]. The simulator 190proposed by Bächer *et al.* solves a linkage system iteratively by starting from known 191 values such as the position of the ground joints and the current position of the actu-192 193 ator arm (which can be determined based on the velocity profile of the motor) and solving for any joints that can be solved with the available information (taking into 194account initial positions of the joint). At every iteration, more joints will be solved 195until at the final iteration where all joints are solved. This approach is illustrated 196in Figure 3. Unlike the numerical solvers, these solvers cannot handle all kinds of 197 198 mechanisms and are limited to dyadic loops only, however, systems with complex kinematic loops either can be converted to equivalent dyadic mechanisms and if they 199 cannot be converted to such mechanisms they must have a non-unique solution or 200 "Branch Defects" which from a design perspective makes them undesirable as their 201 kinematics are not predictable and as such their use in real-world applications require 202special considerations. Therefore, while these approaches are limited to mechanisms 203204 with simple kinematic loops consisting of dyadic loops from a designs perspective they offer a near-complete representation of the design space, furthermore, an advantage 205of using them is that the gradients of simulation can be obtained in a similar man-206 ner which enables gradient-based optimization (e.g., editing existing mechanisms to 207fit certain constraints [3]). We adopt the approach of Bächer *et al.* and discuss the 208 details of our work in the following sections. For a more in-depth view of simulation 209 210 methods, readers are referred to [24].

**3. Methodology.** In this section, we will discuss the details of our methodology for both accelerating simulations and performing gradient-based optimization. First, we will discuss the solver we use and our implementations of it, then we will discuss how we approach the gradient-based optimization and our contributions there.

215**3.1.** Accelerating Graphical Solvers. As mentioned before we adopt the solver proposed by Bächer et al. [3] and introduce some improvements to the im-216 plementation to speed up the simulation process. In this approach, we take any given 217218mechanism with simple kinematic loops and rather than performing a dyadic decomposition [24] to identify four-bar loops, take an iterative approach to find the solutions 219220 by modeling mechanisms as graphs. We can solve for any joint which has two known neighbors (*i.e.*, two joints with currently known positions at a given timestep that 221 have linkages connecting to the node we are trying to solve). To solve for a joint with 2.2.2 two known neighbors, we can use the initial positions of the joints and the current 223 224 positions of the known neighbors to solve for the unknown joint. Take the example

illustrated in Figure 2. In this example, joints 1 and 2 are solved at time step t, given this and the fact that we know the initial positions of the joints we can solve for the angle  $\theta$  the linkage between joints 1 and 3 takes in this time step using the following equation:



FIG. 2. This figure shows the simple case of solving for the angle theta for an unsolved joint 3 using the solutions for joints 1 and 2.

Where  $X_0$  is an  $N \times 2$  matrix of initial positions  $(X_{0,1}, \text{ for example, is a 2D vector}$ indicating the initial position of joint 1) and X is an  $N \times 2$  matrix of the positions of joints at the current timestep. As evident this is simply applying the cosine rule to find the angle the linkage takes. Then once this angle theta is determined we can determine the position of our unknown joint 3 using the following equation:

235 (3.2) 
$$X_3 = X_1 + R(\theta) \frac{(X_2 - X_1) \|X_{0,3} - X_{0,1}\|_2}{\|X_2 - X_1\|_2}$$

Where  $R(\theta)$  is the 2D rotation matrix for a given angle theta. One important thing 236 to note here is that the value of the term inside the inverse cosine function in (3.1)237238can become larger than 1 or smaller than -1 which leads to no solution. This happens when a mechanism is locking or degenerating. Determining the exact timesteps that 239240 the mechanism locks at would require solving massive impractical non-linear equations as such in most cases the practical solution to identifying locking mechanisms is by 241 refining the timesteps of the simulation to ensure that the mechanism does not lock 242at any point. This is one of the reasons why fast solvers are necessary as many 243244high-fidelity simulations are needed just to identify feasible mechanisms.



FIG. 3. Here we illustrate the path the solver takes to find the solution. At first, the solver starts with the known joints (i.e., fixed and actuated joints) and at every step nodes with two known neighbors can be found, in this mechanism illustrated, the path to the solution has 3 steps. The numbers of the joints indicate the order in which the solution is found and the arrows indicate which two neighboring joints are needed to solve the given joint. Known nodes are highlighted in green.

Now that we established how we can solve for joints with two known neighbors 245all that is left to do is to find a path to solving a given mechanism for each timestep. 246 Starting from joints with always-known positions at all timesteps (actuator arm and 247fixed joints), we start by identifying joints with two known neighbors these joints are 248249what we can solve for first at every timestep. Then we repeat the process but with the joints, we identified in the prior step considered as known joints and determine which 250joints can be solved for given the solution to these joints. We repeat this process 251until all the joints in the mechanism have been solved. This process is similar to a 252breadth-first search in graphs and it is visualized for a simple mechanism in Figure 3. 253In doing this a path to the solution of all joints is found which can be used to find 254255solutions at different timesteps. Note that this process only needs to be done once for every mechanism to find the path to the solution and to solve all the timesteps we 256would only traverse this already established path. The process for finding the path 257to the solution is described in Algorithm 3.1. 258

259Once the path to the solution is found we can then run the simulation for all the necessary timesteps. We can perform this task naively by iterating through each 260 timestep separately in a loop, however, as we mentioned in (3.1) the solution to each 261 timestep does not rely on prior timesteps therefore these calculations can be made in 262parallel. As a baseline, we can look at the naive algorithm described in Algorithm 3.2. 263264One obvious way to accelerate this simulator is simply run the timestep for loop in parallel using multiple threads. This will immediately speed up the simulations 265266 significantly depending on the number of threads available. However, using highlevel Julia code to do this will not allow us to optimize the simulation speed to the 267maximum speed it can get to as low-level optimization is necessary for such a thing. 268 The good news is that the underlying linear algebra packages implemented in Julia use 269270 highly optimized code which has been optimized as much as possible over decades of

Algorithm 3.1 Path Algorithm

Require: initialNodes	fixed joints and motor
Require: activeList	
for all $i \in initialNodes do$	
activeList.insert(neighbors(initialNodes(i)))	
end for	
while !activeList.empty() do	
k = activeList.pop front()	
vn = visitedNeighbors(k)	
if vn.size(); 1 then	
i = vn(1), j = vn(2)	
addRule(i,j,k) Add the solution dyadic $(i,j,k)$	$(j) \rightarrow k$ to the list of operation
steps	
assignNextIndex(k), setVisited(k)	
activeList.append(unvisitedNeigbors(k))	
else	
activeList.push $back(k)$	
end if	
end while	

research on matrix and vector operations. Therefore what we need to do to accelerate 271the solver to the maximum extent possible is to vectorize the process of finding the 272solution for all time steps in one go using only vector and matrix operations. In 273this way, we replace the entire for loop in Algorithm 3.2 for timesteps with a single 274function or a few lines of code that compute the solution for all timesteps at once. 275276For the sake of brevity, we do not provide the specific code in the main body of the report but rather include it in Appendix A. By doing this we only use built-in linear 277algebra operations (*i.e.* matrix multiplication and vector operations) which are highly 278optimized in the backend of Julia. In this way, we take advantage of the low-level 279280 optimization done for linear algebra packages and with minimal effort achieve great code efficiency.

Algorithm 3.2 Naive Solver	
<b>Require:</b> $X_0$	initial positions of joints
<b>Require:</b> initialNodes, NTimesteps	
<b>Require:</b> $X = zeros(NTimesteps,NJoints,2)$	
Path $\leftarrow$ Use Algorithm 3.1 to find the path to the solution	ition
for $t \leftarrow 1$ to NTimesteps do	
Compute X[t,initialNodes] fixed joints and actu	ator position at timestep t
for all step $\in$ path do	
$(i,j,k) \leftarrow step$	
$X[t,i] \leftarrow Use (3.1)$ to compute i using j and k	
end for	
end for	
return X	

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Beyond vectorizing the simulation for all timesteps in one mechanism, we can take this vectorization to another level, and vectorize the process for a batch of mech-

anisms. That is to say that we simulate not just for all the timesteps at the same time

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but rather solve for all the timesteps and all the mechanisms in a batch at the same 285 286time. However, to do this we need to take into account a few considerations. Vectorized processes require all the mechanisms to have the same number of joints so as to 287make the matrices and vectors representing all the mechanisms the same size so that 288 we can batch them into higher dimensional arrays and apply built-in linear algebra 289operations in batch. However, different mechanisms can have different sizes, there-290fore, we need to somehow resize the mechanisms without changing their kinematics. 291 Luckily, this is a fairly simple thing to achieve. When batching mechanisms we can 292simply add fixed joints that are not connected to anything, and therefore do not alter 293 the kinematics of the mechanism. Using this approach we simply add as many fixed 294joints to mechanisms in a batch to make sure that they are all the same size (*i.e.*, the 295296 maximum size, and in this work we limit the size to mechanisms with 20 joints). By doing this we make sure that all the mechanisms in a batch are of the same size. As 297it can be seen in Appendix A in the vectorized solver we still compute the path for a 298 mechanism and traverse through the path to the solution, which is different for each 299mechanism. However, if we wish to solve a batch of mechanisms this is not going to 300 be possible as batch operations must be applied at the same time, and this cannot 301 302 be done in different paths simultaneously. Therefore, the only way we can simulate a batch of mechanisms is if they all have the same path to the solution. This however is 303 clearly not the case for different mechanisms, but this does not have to be the case in 304 general. To address this all we have to do is sort the joint ordering for all mechanisms 305 in a batch such that the path to the solution for all mechanisms is to simply start 306 307 with the first joint and compute the solution one joint at a time in order. Therefore, 308 to overcome this second challenge we simply first sort all the mechanisms in a batch as well. For the sake of brevity, we do not go into the algorithm and code details here 309 and include the Julia code for both preprocessing a batch and simulating a batch in 310 Appendix B. 311

At this point, we have discussed the three main ways that we can accelerate the solver; 1) Multi-Threding, 2) Vectorizing over timesteps, and 3) Vectorizing over timesteps and batches of mechanisms. In the sections that follow we will experiment with these approaches and demonstrate that the proposed methods truly do accelerate the simulation of the mechanisms significantly. Furthermore, we will also implement similar solvers that run on the GPU and demonstrate that by utilizing the high throughput of modern GPUs we can further accelerate the simulations.

3.2. Computing Gradient And An Optimization Scheme For Path Syn-319 thesis. So far we have discussed ways to accelerate the solver, however, as we dis-320 cussed in the background section there is great interest in gradient-based optimization 321 322 methods for path synthesis as well. Moreover, the ForwardDiff.jl package in Julia allows us to easily obtain gradients and jacobians of our solver without any additional 323 effort. This means that it is worthwhile to investigate gradient-based optimization us-324 ing our fast solvers as the accelerated solver not only speeds up the forward simulation 325 process it also enables that much faster gradient calculations for the solver, effectively 327 accelerating both simulation and optimization at the same time. However, simply applying ForwardDiff.jl to the solver would not be doing this project justice as there 328 329 are many challenges in the gradient-based optimization of planar linkage mechanisms that are currently unaddressed with the majority of the work in literature focusing on 330 the path synthesis problem from the perspective of design space exploration through 331 genetic algorithms, RL, or deep learning enhanced numerical atlas approaches. The 332 333 main reason these existing methods do not use an optimization-based refinement in



FIG. 4. Here we illustrate the overall gradient decent approach and how we perform this compared to the conventional approach of applying a weighted penalty to the objective.

334 their work is because of a few notable challenges which have yet to be addressed by researchers in the community. In this project, we aim to address three major chal-335 lenges in gradient-based optimization for path synthesis which make the application of 336 gradient-based optimization rather difficult. One of the challenges is slow solvers and 337 gradient calculations which make the iterative process of optimization rather slow. 338 339 This challenge is already addressed thanks to our significantly faster solver. However, two other challenges exist that we still need to address. These issues are the prob-340 lem of picking a good objective function for comparing target paths and traced paths 341 and the other is how to prevent mechanisms to fall into locking configurations during 342 optimization. Both of these matters boil down to obtaining a reasonable objective 343 function to allow us to match our target path as well as possible while still remaining 344 345 in the feasible space. We will discuss the details of our implementation in the sections that follow. 346

347 **3.3. Shape Matching Objective.** The main objective of path synthesis is to generate mechanisms that trace a path that matches any arbitrary path with any 348 arbitrary number of points. This means that if we simulate a mechanism for say 349 200 timesteps but our objective curve is a hand-drawn path with only a handful of 350 351 points we cannot simply compare the points of the path generated by a mechanism and our target directly. Furthermore, the simulated curve has information regarding 352 velocity, that is to say, if the mechanism is moving faster in a given part of the path 353 the distance between the points will be larger, however, in path synthesis the only 354 objective is to match a given target curve purely from a geometric perspective without 356 any dynamic considerations. In fact, in practice, the dynamics can be controlled by adjusting the speed of the actuator as needed. As such, we need to come up with an 357 358 objective function that unlike the work by Bächer *et al.* [3] does not rely on direct editing of coupler path already traced by a machine but rather is generalized to any 359 arbitrary shape we wish to optimize for. This can be done using single-directional or 360 bi-directional chamfer distance. The general form of the equation for chamfer distance 361 362 between two sets of points  $S_1$  and  $S_2$  is as follows:

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363 (3.3) 
$$d_{CD}(S_1, S_2) = \sum_{x \in S_1} \min_{y \in S_2} \|x - y\|_2^2 + \sum_{y \in S_2} \min_{x \in S_1} \|x - y\|_2^2$$

Where the first term measures the distance from  $S_1$  to  $S_2$  and the second term the 364 365 other way around. As such this is usually referred to as the bi-directional chamfer distance. However, in path synthesis, we may be interested in only capturing the desired 366 path as part of the mechanism's output curve. That is because even if the mechanism 367 368 only traces the desired path in part of its motion the problem of synthesizing that path is solved. Therefore, in some instances, it may make sense to include only the 369 370 term that measures the distance from the target curve to the traced path. However, for the results and discussions presented in this report, we will use the bi-directional 371 chamfer distance and not make any further modifications to this objective function. 372

**3.4.** Preventing Locking In Mechanisms. As we saw in (3.1) the solution 373to any of the joints in the mechanisms requires the computation of an inverse cosine, 374 and if a mechanism is set up such that during the motion of the actuator at some 375 376 timestep the value inside the inverse cosine function is above 1 or below -1, the simulation fails, which indicates that the mechanism locks in at time between the last 377 timestep and the timestep without a solution. This presents a challenge in gradient-378 based optimization as now a highly non-linear and complex constraint is added to the 379 optimization problem which if ignored will easily render gradient-based optimization 380 381 useless. Furthermore, the complexity of the constraint is beyond its non-linear nature, 382 as the algorithm for the solution traverses a specific path through the graph representing the mechanisms. This means that if at one point in the path to the solution, 383 a locking joint is identified the steps in the solution path after this point also cannot 384 be solved for since they rely on the solution of prior joints in the path to be solved. 385This means that even if we can measure how much the cosine value of a given joint 386 387 that locking is above 1 or below -1 we cannot say anything about how using gradients of this constraint violation for healing this issue will affect the solution of all the other 388 joints. Therefore, a straightforward measure of constraint violation in cases of failure 389 is not practical in this case. However, we may yet be able to perform gradient-based 390 optimization by adding a penalty term to the objective of the optimization. Looking 391 at prior works Bächer et al. [3] propose a penalty strategy by adding an objective to 392 393 increase what the authors call distance to locking. This metric is measured by the following equation: 394

395 (3.4) 
$$d_i \left( \mathbf{x}_i, \mathbf{x}_j, \mathbf{x}_k \right) = 1 - \cos \left( \theta \left( \mathbf{x}_i, \mathbf{x}_j, \mathbf{x}_k \right) \right)^2$$

where  $\cos(\theta(\mathbf{x}_i, \mathbf{x}_j, \mathbf{x}_k))$  is the value of the cosine of the angle calculated for joint i using the solutions of joints j and k using (3.1). To prevent this penalty from becoming too restrictive the authors use a cross-entropy penalty with a threshold  $\varepsilon$ and define the penalty objective as:

400 (3.5) 
$$f_i(\mathbf{x}_i, \mathbf{x}_j, \mathbf{x}_k) = -\frac{1}{2}\log^2\left(\frac{1}{\varepsilon}d_i(\mathbf{x}_i, \mathbf{x}_j, \mathbf{x}_k)\right)$$

by doing this they prevent the penalty from being too restrictive until the distance to locking dips below a specific threshold  $\varepsilon$ . However, this penalty will practically 403 encourage the optimization to specifically achieve a distance to locking equal to  $\varepsilon$ . 404 And this is not exactly optimal. Furthermore, we still have to choose a fairly large  $\varepsilon$ . 405 Bächer *et al.* [3] simply add the sum of this equation for all joints to their objective 406 for minimization with some weight applied to it with respect to the main objective 407 and solve for the minimal value of the sum of the two objectives in other words to 408 minimize the following optimization problem:

409 (3.6) 
$$\Delta \overline{X_0} = \arg\min_{\Delta \tilde{X_0}} d_{CD}(S_{target}, X_0) + \lambda \sum_i f_i(X_0)$$

Where  $d_{CD}(S_{target}, X_0)$  is the chamfer distance between the path traced by a 410 mechanism with initial joint positions  $X_0$  and the target curve with points in the 411 set  $S_{target}$  and  $f_i(X_0)$  is the value of cross-entropy penalty function for joint i in the 412 mechanism with initial joint positions  $X_0$  and  $\lambda$  is the weight for the penalty objective 413 and  $\Delta \overline{X_0}$  is the optimal changes applied to the initial joint positions  $X_0$  to optimize 414 the mechanism for path synthesis. In the end, although this penalty approach makes 415 the optimization process clear-cut, it still limits the path synthesis potential with 416 optimization as it will always directly work against the main objective of reducing 417 chamfer distance. As such we introduce an alternative to this by suggesting a different 418 419 strategy for preventing locking, which is similar in nature to this approach but gives the optimization more freedom. We propose using simple gradient descent with a step 420 size of  $\alpha$  however instead of just optimizing for minimal chamfer distance we linearly 421 orthogonalize the chamfer distance gradient with respect to distance to locking. By 422 doing this we make an assumption that for small enough  $\alpha$  the objective can be 423 424 deemed linear, hence making it so that if we orthogonalize the gradient of chamfer distance with respect to distance to locking we move only in the direction that reduces 425 chamfer distance without changing the distance to locking for the mechanism. This 426 assumption of course in practice is not going to be accurate as we still have to choose 427 a sufficiently large  $\alpha$  to make the optimization practical, however by doing this we 428 429 ensure that if the distance to locking is to increase during the optimization the rate at which it will happen during the gradient descent is minimal. Therefore, the update 430 rule we propose for the gradient descent can be described as: 431

432 (3.7) 
$$X_0^{k+1} = X_0^k - \alpha \left( \frac{\partial CD}{\partial X_0} - \frac{\frac{\partial DL}{\partial X_0} \cdot \frac{\partial CD}{\partial X_0}}{\left| \frac{\partial DL}{\partial X_0} \right|^2} \frac{\partial DL}{\partial X_0} \right)$$

433 Where  $CD = d_{CD}(S_{target}, X_0)$  and  $DL = \sum_i f_i(X_0)$  and  $X_0^k$  is the initial joint 434 positions for the mechanism after k steps of gradient decent. We will later demonstrate 435 how this approach is effective for refining mechanisms for path synthesis. The overall 436 workflow of our approach is illustrated in Figure 4.

**4. Results & Discussion.** In this section, we run experiments on different variants of the solver and demonstrate how the approaches we have developed for accelerating the solver actually significantly improve the speed of the solver to a noticeable extent. Finally, we will demonstrate the gradient-based optimization results for a few examples to show the efficacy of the proposed approach for the refinement of mechanisms.

4.1. Acceleration Of The Solver. To measure how well each of the proposed 443 444implementations of the solver performs we conduct an experiment for simulating 10,000 mechanisms with 6 to 20 joints in them using each of the solvers described 445 in prior sections. We solve all of these mechanisms for 200 timesteps and with the 446 actuator moving at constant velocity. We also make a GPU implementation of the 447 batch solver and test the simulation speeds on the GPU as well. As for hardware 448 we use an Intel i9-12900K processor for the CPU versions of the solver and an RTX 449 3090Ti for the GPU implementations of the solver. Furthermore, it is important to 450note that we were unable to get the GPU version to work properly on Julia, as such we 451implemented the GPU versions on Python using Pytorch, and the results presented 452here for the GPU version of the solver are based on experiments on python. The 453454 results of our experiments are presented in Figure 5.



Average Time Taken To Simulate 10,000 Mechanisms

FIG. 5. The results of running the solvers for 10,000 mechanisms. The time reported is the average across 10 runs of each solver.

455 As evident in Figure 5 we see that the naive solver is the slowest solver. However, we can see that a simple approach of multi-threading does not really yield a significant 456 improvement in the results and only accelerates the process by slightly more than 457 double despite the i9-12900k's 24 threads. This is the main reason it is important 458 for us to utilize the lower-level optimized linear algebra packages to maximize the 459460performance of the solver. As evident the vectorized solver is more than 10 times faster than the naive solver and 5 times faster than the multi-threaded solver. Interestingly 461 462 the batch-vectorized solver actually ends up being slower than the multi-threaded solver on the CPU. This is simply because we resize all of the mechanisms to the 463 maximum size of 20 joints which essentially makes the overall solver much slower 464 as the CPU simply does not have enough throughput to truly perform all of the 465466 computations at once despite the highly optimized linear algebra packages. However,

the same cannot be said of the GPU. As it is shown in Figure 5, the GPU can simulate 467 all 10,000 mechanisms using the batch vectorized solver in roughly 100 milliseconds, 468 which is a mindblowing 500 times faster than the naive solver and 50 times faster 469 than the vectorized solver on the CPU. This is simply made possible by the immense 470 throughput of the GPU and the highly optimized CUDA libraries that Pytorch uses 471 in its backend. Furthermore, we see that despite the increased cost of the batch solver 472 resizing all mechanisms to the maximum size the GPU has the necessary throughput 473 to handle the larger number of computations and perform all the simulations in one 474 go. One thing to note is that when we tested the GPU using just a timestep vectorized 475solver the results we got were much slower as each simulation had to be done on the 476 GPU one at a time, despite the GPU having the capacity for much more, which despite 477 478 of the better efficiency of the vectorized solver (as we saw in the CPU) led to slower results overall, demonstrating the importance of developing the batch vectorization 479for a high throughput hardware like the GPU. In conclusion, we see that utilizing 480 the GPU can provide up to 500 times faster simulations and speed up most current 481 approaches employed by researchers for path synthesis to a great extent. 482



FIG. 6. The 5 case studies for demonstrating the effectiveness of gradient-based optimization. Here we visualize the mechanism and their current output curves and the target curves we wish to achieve using these mechanisms through gradient-based optimization.



FIG. 7. The results of gradient-based optimization on the 5 case studies demonstrating the effectiveness of gradient-based optimization. Here we see that our approach of orthogonalization instead of weighted penalty has led to much better results compared to conventional optimization.

TABLE 1

Quantitave results (final chamfer distance) of the optimization comparing both methods is each case-study

Method	Study 1	Study 2	Study 3	Study 4	Study 5	
Before Optimization	0.8155	0.5425	0.2955	0.5051	0.7207	
Conventional	0.6307	0.3703	0.0215	0.3001	0.1056	
Ours	0.4948	0.0049	0.0065	0.2909	0.1648	

4.2. Experiments On Gradient-Based Optimization. To demonstrate the 483 effects of the gradient-based optimization approach we propose we conduct a case 484 study with 5 mechanisms and 5 target paths which for each mechanism. These mech-485anisms and the test targets for each case study can be seen in Figure 6. We then run 486 gradient descent using our method and the naive method with a constant penalty with 487 weight lambda = 0.1 (see (3.6)) for 10,000 steps in each case-study with a gradient 488 decent step size of 0.0001 and compare the results visually in Figure 7 and report the 489 minimum chamfer distance achieved by each method in Table 1. 490

The first thing we see visually is that in all cases the outputs of the mechanisms 491 492have improved significantly with the exception of the first case study where both conventional and our optimization have failed to improve the output of the mechanism. 493 However, in all other cases, we can both visually (Figure 7) and quantitatively (Ta-494 ble 1) confirm the improvements. Specifically in the second, third, and last case 495studies, we see amazing improvements in the mechanisms matching their targets with 496497great accuracy. However, as it is clear the orthogonalization has led to much better outcomes (the only exception is the last case study) with the mechanisms while 498499 the conventional approach has mostly failed to provide significant enough improvements in the mechanisms. This shows that unless the optimization is being done 500 on a machine that traces a path that is already very close to the desired path the 501 conventional approach is simply not good enough and does not provide a worthwhile 502503 refinement, which is likely why most researchers have not applied such methods. But we showed that with our improved method, we are able to do much better and prove that refinement through optimization is a viable option for path synthesis.

5. Conclusion. In this project, we set out to address one of the major challenges 506 that has slowed progress in inverse kinematics which is a lack of very fast solvers for 507 508 linkage mechanisms. As we discussed the faster these solvers are the better existing methods for inverse kinematic synthesis can get. Given this, we developed very fast 509 vectorized solvers for linkage mechanisms that can enable up to 500 times faster simu-510lations when GPUs are utilized, which can improve the speed of existing optimization 511512and data-driven approaches for inverse kinematic design. Furthermore, we took advantage of the ForwardDiff.jl package in Julia to develop an improved gradient-based 513optimization method that can outperform the existing methods for gradient-based op-514timization for path synthesis, and we demonstrated the efficacy of our method and its 515superiority to existing methods through a case study of 5 optimization problems. In 516conclusion, in this project, we were able to accelerate planar linkage mechanism simulations by up to 500 times while also improving gradient-based optimization methods 518519 for path synthesis using the automatic differentiation of our fast solvers. We hope that this contribution will help accelerate progress in the field of inverse kinematic 520 design and path synthesis. 521

522 Finally, we provide the code used in this project publicly which can be found at 523 https://github.com/ahnobari/18337-Linakge-Project.

524 **Appendix A. Julia Code For Vectoized Solver.** Below is the Julia code 525 to traverse the path to the solution and compute all the timesteps at once.



FIG. 8. Vectorized solver to solve for all timesteps at once.

526 **Appendix B. Julia Code For Batch Vectoized Solver.** Below is the code 527 for preprocessing a list of mechanisms into a batch for the batch solver. This part 528 includes the sorting and resizing of the batch.



FIG. 9. Preprocessing code for the batch solver.

529 Once the batches have been prepared the code below is used to solve not just all 530 timesteps but all the mechanisms in the batch as well:

nction batch_solve(Cs,x0s,Gs,node_types,thetas)
<pre>x = zeros(size(x0s)[1],size(x0s)[2],size(thetas)[1],2) x = x .+ reshape(x0s .* node_types,size(x0s)[1],size(x0s)[2],1,2)</pre>
x[:,2,:,:] = x[:,1,:,:] + repeat(reshape([cos.(thetas) sin.(thetas)],1,size(thetas)[1],2),size(x0s)[1],1,1) .* Gs[:,1,2]
<pre>t = zeros(size(x0s)[1],size(x0s)[2],size(x0s)[2]) t[:,:,[1,2]] .= 1</pre>
t = t .* node_types
Cs += t
<pre>for k in 3:size(x0s)[2]     inds = findall(x-&gt; x&gt;0 , Cs[:,k,1:k])     x_vals = x[inds,:,:]</pre>
<pre>xis = x_vals[1:size(x0s)[1],:,:] xjs = x_vals[size(x0s)[1]+1:2*size(x0s)[1],:,:]</pre>
l_ijs = xis - xjs l_ijs = 1_ijs .^ 2 l_ijs = sum(l_ijs,dims=3) l_ijs = sqrt.(l_ijs)
<pre>g_ij_k = Gs[inds,k] g_ik = g_ij_k[1:size(x0s)[1],:,:] g_jk = g_ij_k[size(x0s)[1]+:2*size(x0s)[1],:,:]</pre>
cosphis = (l_ijs.^2 .+ g_ik.^2 g_jk.^2)./(2 * l_ijs .* g_ik)
<pre>singularities = findall(x-&gt;x&gt;1.0    x&lt;-1.0, cosphis)</pre>
cosphis[singularities] .= 0.0
x0_val = x0s[inds,:]
<pre>x0i1 = x0_val[1:size(x0s)[1],2] x0i0 = x0_val[1:size(x0s)[1],1]</pre>
x0j1 = x0_val[size(x0s)[1]+1:2*size(x0s)[1],2] x0j0 = x0_val[size(x0s)[1]+1:2*size(x0s)[1],1]
x0k1 = x0s[:,k,2] x0k0 = x0s[:,k,1]
s = sign.((x0i1-x0k1).*(x0i0-x0j0) - (x0i1-x0j1).*(x0i0-x0k0))
<pre>phi = s .* acos.(cosphis)</pre>
R = reshape([[cos.(phi) sin.(phi)] [-sin.(phi) cos.(phi)]],size(x0s)[1],200,2,2)
scaled_ij = (xjs-xis)./l_ijs .* g_ik
<pre>x_k = xis + reshape(permutedims(permutedims(reshape(R,size(thetas)[1]*size(x0s)[1],2,2),[2,3,1]) =</pre>
x[:,k,:,:] += x_k
end
return permutedims(x,[1,3,2,4]

FIG. 10. Vectorized solver to solve for all timesteps and mechanisms in the batch at once.

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### REFERENCES

- [1] On the Extension of a Fourier Descriptor Based Method for Four-Bar Linkage Synthesis for Generation of Open and Closed Paths, vol. Volume 2: 34th Annual Mechanisms and Robotics Conference, Parts A and B of International Design Engineering Technical Conferences and Computers and Information in Engineering Conference, 08 2010, https://doi.org/10.1115/DETC2010-29028, https://doi.org/
  10.1115/DETC2010-29028, https://arxiv.org/abs/https://asmedigitalcollection.asme.org/
  IDETC-CIE/proceedings-pdf/IDETC-CIE2010/44106/923/4549582/923\_1.pdf.
- 539 [2] Kinematic Synthesis Using Reinforcement Learning, vol. Volume 2A: 44th Design
   540 Automation Conference of International Design Engineering Technical Confer-

541		ences and Computers and Information in Engineering Conference, 08 2018, https:
542		//doi.org/10.1115/DETC2018-85529, https://doi.org/10.1115/DETC2018-85529, https:
543		//arxiv.org/abs/https://asmedigitalcollection.asme.org/IDETC-CIE/proceedings-pdf/
544		eq:IDETC-CIE2018/51753/V02AT03A009/2475681/v02at03a009-detc2018-85529.pdf.
545		V02AT03A009.
546	[3] M	. BÄCHER, S. COROS, AND B. THOMASZEWSKI, Linkedit: Interactive linkage editing using
547		symbolic kinematics, ACM Trans. Graph., 34 (2015), https://doi.org/10.1145/2766985,
548		https://doi-org.libproxy.mit.edu/10.1145/2766985.
549	[4] J.	CABRERA, A. SIMON, AND M. PRADO, Optimal synthesis of mechanisms with genetic al-
550		gorithms, Mechanism and Machine Theory, 37 (2002), pp. 1165–1177, https://doi.org/
551		https://doi.org/10.1016/S0094-114X(02)00051-4. https://www.sciencedirect.com/science/
552		article/pii/S0094114X02000514
553	[5] J	CHU AND I SUN A New Approach to Dimension Symphosis of Spatial Four-Bar
554	[0] 0.	Linkage Through Numerical Atlas Method Journal of Mechanisms and Robotics
555		2 (2010) https://doi.org/10.1115/1.4001774
556		2 (2015), https://doi.org/10.1110/1.4001/44, https://doi.org/no.1110/1.4001/44,
557		ntips://axiv.org/abs/ntips://asinetigitaiconection.asine.org/inechanisiis/obotics/
EEO	[c] I	attick-put/2/4/041004/0512361/041004_1.put. 041004.
550	[0] J.	CHU AND J. SUN, Numerical allas method for pain generation of spherical four-oar
559		mechanism, Mechanism and Machine Theory, 45 (2010), pp. 867–879, https://doi.org/
560		https://doi.org/10.1016/j.mechmachtheory.2009.12.005, https://www.sciencedirect.com/
561	[ <b>-</b> ] a	science/article/pii/S0094114X09002286.
562	[7] S.	DESHPANDE AND A. PURWAR, A Machine Learning Approach to Kinematic Synthe-
563		sis of Defect-Free Planar Four-Bar Linkages, Journal of Computing and Informa-
564		tion Science in Engineering, 19 (2019), https://doi.org/10.1115/1.4042325, https://
565		doi.org/10.1115/1.4042325, https://arxiv.org/abs/https://asmedigitalcollection.asme.org/
566		$computing engineering/article-pdf/19/2/021004/5998446/jcise_{019_{-}02_{-}021004,pdf.\ 021004.$
567	[8] S.	DESHPANDE AND A. PURWAR, Computational Creativity Via Assisted Variational
568		Synthesis of Mechanisms Using Deep Generative Models, Journal of Mechani-
569		cal Design, 141 (2019), https://doi.org/10.1115/1.4044396, https://doi.org/10.1115/1.
570		4044396, https://arxiv.org/abs/https://asmedigitalcollection.asme.org/mechanicaldesign/
571		article-pdf/141/12/121402/5874716/md_141_12_121402.pdf. 121402.
572	[9] S.	DESHPANDE AND A. PURWAR, An Image-Based Approach to Variational Path Syn-
573		thesis of Linkages, Journal of Computing and Information Science in Engineering,
574		21 (2020), https://doi.org/10.1115/1.4048422, https://doi.org/10.1115/1.4048422,
575		https://arxiv.org/abs/https://asmedigitalcollection.asme.org/computingengineering/
576		article-pdf/21/2/021005/6577132/jcise_21_2_021005.pdf. 021005.
577	[10] S.	DESHPANDE AND A. PURWAR, An image-based approach to variational path synthesis of
578		linkages, Journal of Computing and Information Science in Engineering, 21 (2021).
579	[11] S.	EBRAHIMI AND P. PAYVANDY, Efficient constrained sunthesis of path generating four-bar
580	L ]	mechanisms based on the heuristic optimization algorithms. Mechanism and Machine The-
581		ory, 85 (2015), pp. 189–204, https://doi.org/https://doi.org/10.1016/j.mechmachtheory,
582		2014 11 021, https://www.sciencedirect.com/science/article/pii/S0094114X14003036
583	[12] M	B FOGELSON C TUCKER AND I CAGAN GCP-HOLO: Generating High-
584	[12] 11	Order Linkage Graphs for Path Simthesis Journal of Mechanical Design 145
585		(2023) https://doi.org/10.1115/1.4062147 https://doi.org/10.1115/1.4062147
586		https://arviv.org/abs/https://amedigitalcollection_asme_org/mechanicaldesign/
587		article-pdf/145/7/073303/7001642/md 145 7 073303 pdf 073303
588	[13] N	a total pull 140/14013030/1001042/interferencesional synthesis of mechanical linkages using
580	[10] 14	artificial neural networks and fourier descriptors. Machinesis of mechanical sciences 6 (2015), pp. 20–24
500		artificial neural networks and joiner descriptors, Mechanical Selectes, 0 (2015), pp. 25-54, https://doi.org/10.5104/ms.6.20.2015.https://ms.conomicus.org/articles/6/20/2015/
501	[14] N	KuAN I ULIA AND M ALCOALD Dimensional surthasis of machanical linkages using
502	[14] 10	artificial neural networks and fourier description Machines of Mechanical interaction of 200.24
502	[15] D	artificial neural networks and journet associations, Mechanical Sciences, 0 (2013), pp. 29–34.
595	[15] D	- F. KINGMA AND M. WELLING, Auto-encound variational bayes, 2014, https://arxiv.org/
505 505	[1 <i>6</i> ] II	aus/1012.0114. LIDGON A Delegation Method for Simulating the Virgensting of Community
595	[10] H	Nuclinear Mechanisma Learnel of Mechanical Dr. 100 (2005) 710
596		<i>Nonunear Mechanisms</i> , Journal of Mechanical Design, 128 (2005), pp. 719–
097		(20, nttps://doi.org/10.1115/1.2198255, nttps://doi.org/10.1115/1.2198255,
598		nttps://arxiv.org/abs/nttps://asmedigitalcollection.asme.org/mechanicaldesign/
599	[4 =] **	article-pdf/128/4/(19/5923963/(19_1.pdf.
600	[17] H	LIPSON, Evolutionary synthesis of kinematic mechanisms, Artificial Intelligence for Engi-
601		neering Design, Analysis and Manufacturing, 22 (2008), p. 195–205, https://doi.org/10.
602		1017/50890060408000139.

- [18] J. M. MCCARTHY AND G. S. SOH, Geometric design of linkages, vol. 11, Springer Science &
   Business Media, 2010.
- [19] J. R. MCGARVA, Rapid search and selection of path generating mechanisms from a library, Mechanism and Machine Theory, 29 (1994), pp. 223-235, https://doi.org/
   https://doi.org/10.1016/0094-114X(94)90032-9, https://www.sciencedirect.com/science/
   article/pii/0094114X94900329.
- [20] V. MNIH, K. KAVUKCUOGLU, D. SILVER, A. GRAVES, I. ANTONOGLOU, D. WIERSTRA, AND
   M. RIEDMILLER, Playing atari with deep reinforcement learning, 2013, https://arxiv.org/
   abs/1312.5602.
- [21] P. RADHAKRISHNAN AND M. I. CAMPBELL, A graph grammar based scheme for generating and
   evaluating planar mechanisms, in Design Computing and Cognition '10, J. S. Gero, ed.,
   Dordrecht, 2011, Springer Netherlands, pp. 663–679.
- [22] L. REGENWETTER, A. H. NOBARI, AND F. AHMED, Deep generative models in engineering
   design: A review, CoRR, abs/2110.10863 (2021), https://arxiv.org/abs/2110.10863, https:
   //arxiv.org/abs/2110.10863.
- 618 [23] F. REULEAUX, Lehrbuch der Kinematik, vol. 1, Vieweg, 1875.
- [24] S. SHARMA AND A. PURWAR, Using a Point-Line-Plane Representation for Unified Simulation of Planar and Spherical Mechanisms, Journal of Computing and Information Science in Engineering, 20 (2020), https://doi.org/10.1115/1.4046817, https://
  doi.org/10.1115/1.4046817, https://arxiv.org/abs/https://asmedigitalcollection.asme.org/ computingengineering/article-pdf/20/6/061002/6537582/jcise\_20\_6\_061002.pdf. 061002.
- [25] K. SOHN, H. LEE, AND X. YAN, Learning structured output representation using deep conditional generative models, in Advances in Neural Information Processing Systems, C. Cortes, N. Lawrence, D. Lee, M. Sugiyama, and R. Garnett, eds., vol. 28, Curran Associates, Inc., 2015, https://proceedings.neurips.cc/paper/2015/file/ 8d55a249e6baa5c06772297520da2051-Paper.pdf.
- [26] J. SUN, H. LU, AND J. CHU, Variable step-size numerical atlas method for path generation of spherical four-bar crank-slider mechanism, Inverse Problems in Science and Engineering, 23 (2015), pp. 256–276, https://doi.org/10.1080/17415977.2014.890615, https: //doi.org/10.1080/17415977.2014.890615, https://arxiv.org/abs/https://doi.org/10.1080/
  17415977.2014.890615.
- [27] I. ULLAH AND S. KOTA, Optimal Synthesis of Mechanisms for Path Generation Using Fourier Descriptors and Global Search Methods, Journal of Mechanical Design, 119 (1997), pp. 504–510, https://doi.org/10.1115/1.2826396, https://doi.org/10.1115/1.
  2826396, https://arxiv.org/abs/https://asmedigitalcollection.asme.org/mechanicaldesign/ article-pdf/119/4/504/5600257/504\_1.pdf.
- [28] S. VAREDI-KOULAEI AND H. REZAGHOLIZADEH, Synthesis of the four-bar linkage as path generation by choosing the shape of the connecting rod, Proceedings of the Institution of Mechanical Engineers, Part C: Journal of Mechanical Engineering Science, 234 (2020), pp. 2643-2652, https://doi.org/10.1177/0954406220908616, https://doi.org/10.1177/ 0954406220908616, https://arxiv.org/abs/https://doi.org/10.1177/0954406220908616.
- [29] A. VASILIU AND B. YANNOU, Dimensional synthesis of planar mechanisms using neural net works: Application to path generator linkages, Mechanism and Machine Theory, 36 (2001),
   pp. 299–310, https://doi.org/10.1016/S0094-114X(00)00037-9.
- [30] K. J. WALDRON AND S. V. SREENIVASAN, A Study of the Solvability of the Position Problem for Multi-Circuit Mechanisms by Way of Example of the Double Butterfly Linkage, Journal of Mechanical Design, 118 (1996), pp. 390–395, https://doi.org/10.1115/1.2826898, https:// doi.org/10.1115/1.2826898, https://arxiv.org/abs/https://asmedigitalcollection.asme.org/ mechanicaldesign/article-pdf/118/3/390/5747759/390\_1.pdf.
- [31] J. WU, Q. J. GE, F. GAO, AND W. Z. GUO, On the Extension of a Fourier Descriptor
  Based Method for Planar Four-Bar Linkage Synthesis for Generation of Open and Closed
  Paths, Journal of Mechanisms and Robotics, 3 (2011), https://doi.org/10.1115/1.4004227, https://doi.org/10.1115/1.4004227, https://arxiv.org/abs/https://asmedigitalcollection.
  asme.org/mechanismsrobotics/article-pdf/3/3/031002/5590355/031002\_1.pdf. 031002.